

The Relationship between Force and Time with Lagrange Formula by Regulating Piston Mass on Crank Linkage of Vehicle

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Abstract

In this study the simulation about force and time is built with Lagrange equation which solves the crankshaft dynamics in engine of vehicle. The force will incline as the time inclines with a certain mass of piston of 1.6Kg, 3.2Kg & 4.8Kg in engine on vehicle, meantime as the mass inclines the force will incline too from 16N and 32N to 48N with maximum value at 7E-3s at 700r/m rotation of engine. The bad force will be formed in first and then the good one will be followed at last in terms of former $d^2F/dt^2 < 0$ and $d^2F/dt^2 > 0$ respectively. But the bad force has good stability. If the mass of piston is increased the bigger force may be formed in engine.

Keywords: Relationship; Force And Time; Lagrange Equation; Crank Shaft; Crank; Crank Linkage; Vehicle.

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INTRODUCTION

The crank shaft is an important mechanism in vehicle which includes crank, crank linkage and piston on engine so the research will be preceded on them is necessary method to calculate with modeling in recent study. So this paper will search the detail database to establish the equation to solve it further. [1-3] the dynamics can solve the key problem of intrinsic relationship between force and time in engine which can express the detail behavior to the crank shaft force analysis for us to find intrinsic things.

As recent the vehicle has been grown rapidly in world so the most significant engine part of crank shaft will play more and more roles in future. The fatigue life will be key to its span life so the force change with time of rotation will be important data for us to search deeply. The force behavior must be established to further clarify the fatigue role and play a key role. Since the time limits the fatigue life wouldn't be searched here we only play to establish dynamic modelling in terms of Lagrange equation to crank shaft mechanism. We try to find role of force formed on piston mass at certain rotation speed and time in order to find its maximum force and its stability.

We want to find the mass condition which causes the bigger force for engine of crankshaft so it is

searched on its status in this study. On the other side the shaft rotation is studied too to look for the appropriate effects.

Modeling and Calculation

According to Figure 1 and 2 which is kinematic graphs on the crankshaft in engine in vehicle, From Figure 2 it is supposes that crank $R=60\text{mm}$, crank linkage $L=210\text{mm}$. This is the engine driving crankshaft, $\Delta l=l_{\text{max}}-l_{\text{min}}$. A is piston and cylinder wall; v is its speed; n is shaft rotation.

It has

$$\Delta l = \sqrt{L^2 + R^2 - 2LR \cos \left[\pi - \arcsin \left(\frac{R}{L} \sin \theta_1 \right) - \theta_1 \right]} \dots \dots \dots (1)$$

Here $\theta_1 = 2\pi n t / 60$.

According to Lagrange equation

$$\frac{d}{dt} \left(\frac{\partial E_k}{\partial \dot{q}_i} \right) - \frac{\partial E_k}{\partial q_i} + \frac{\partial E_p}{\partial q_i} = F_i \dots \dots \dots (2)$$

The equation (2) is the crank shaft dynamic equation in terms of Figure 2. Here rotary inertia is

$$I_1 = \frac{1}{3} m_1 l_1^2 ; I_2 = \frac{1}{3} m_2 l_2^2 \dots \dots \dots (3)$$

It is supposed that $m_1=0.6\text{Kg}$; $m_2=1\text{Kg}$; $m_3=0.25\text{Kg}$.
 And $\omega_1 = \dot{\theta}_1$ (4)
 So it has $\omega_1 = t^2 \theta_1$ (5)

The same as above it has
 $\omega_2 = t^2 \theta_2$ (6)
 $\omega_3 = t^2 \theta_3$ (7)
 And $v_3 = t^2 [\Delta l - (L-R)]$ (8)

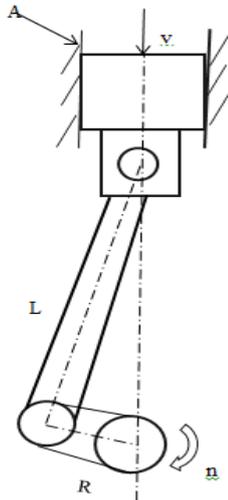


Fig-1: The kinematic of crankshaft linkage length in the engine of vehicle.

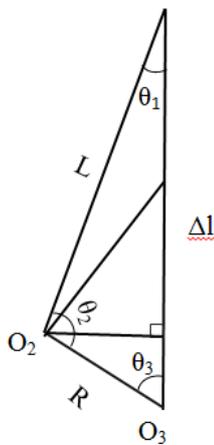


Fig-2: The kinematic of crankshaft linkage mechanism in engine of vehicle.

DISCUSSIONS

In Fig-3 the curve will be separated two parts where one is $d^2F/dt^2 > 0$ as seen in right which expresses it has the minimum point and the other is $d^2F/dt^2 < 0$ in left which expresses that it has maximum one. That said that the steep inclining first and the sluggish inclining at last and it expresses that the big force is formed first in a cycle so it has been paid attention to the big force which may cause grinding heavily in first angles. The force will incline as the time is big. With the maximum 0~1.6Kg distribution will cause the small force 16N too. In Figure 4 and 5 the curve will incline the force to

32N and 48N respectively with the 0~3.2Kg and 0~4.8Kg at the 7E-3s and 700r/m.

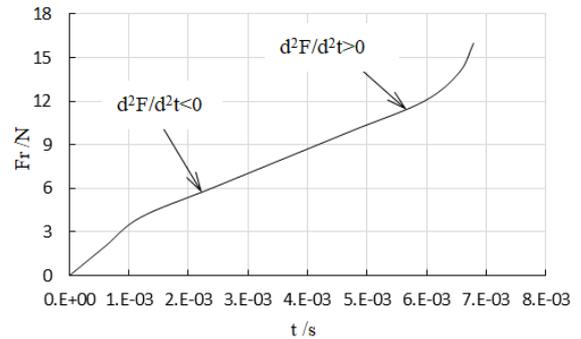


Fig-3: The relationship between force and time in crank shaft with rotation of 700r/m and 0-1.6Kg mass piston on vehicle.

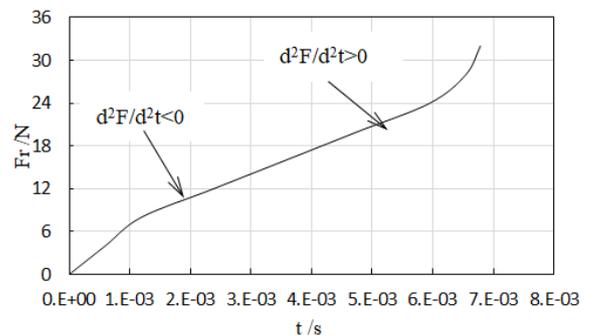


Fig-4: The relationship between force and time in crank shaft with the rotation of 700r/m and regulating from 0 to 3.2Kg mass piston on vehicle.

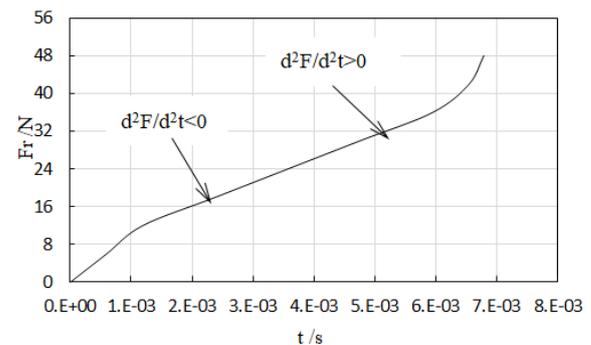


Fig-5: The relationship between force and time in crank shaft with the rotation of 700r/m and regulating from 0 to 4.8Kg mass piston on vehicle.

Fig-5: shows the curve between force and time under $n=700\text{r/m}$. This indicates that the maximum force in the engine is up to 48N ie. 5KgF and the force inclines steep first and inclines in engine with the increase of time. It is no relation to the crank length $R=60\text{mm}$ and linkage $L=210\text{mm}$ in terms of supposed conditions with the calculation result. The periodic time is 7E-3s in 700r/m, it means that the periodic time is the same. Figure 5 shows that the force becomes sinusoidal

likely from 0 to 7E-3s which is the same as $\theta_2=0\sim57^\circ$. The size is larger than the one at 3.2Kg. And 1.6Kg. The force is larger than the above two. The force is used instantaneous time within one cycle to gain the simulation results.

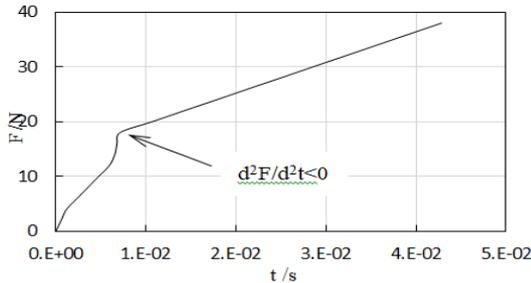


Fig-6: The relationship between force and time in crank shaft with the rotation of 700r/m and regulating from 0 to 3.8Kg mass piston on vehicle.

In Fig-6 the force will be the same as the same as Figure 3 in one cycle of crank angle. It is found that the inclining trend may be gained in this cycle. The maximum force can be beyond 3.5Kg at the end of 4.3E-2s. The same trends with this figure are gained in Figure 7&8 except that the inclining force will be formed here. For example the biggest one will attain 75N and 110N at the end of 4.3E-2s. It expresses that the force with mass piston inclining to 7.6Kg& 11.4Kg may incline too. The turn points in these figures range from 18N to 37N, 55N which explains that the force will be inclined two times.

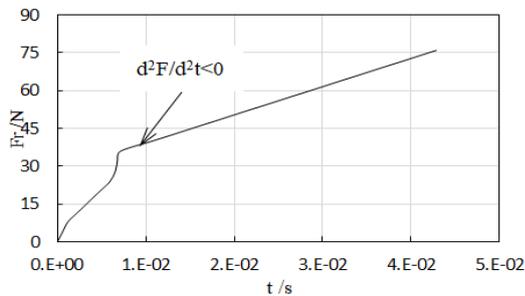


Fig-7: The relationship between force and time in crank shaft with the rotation of 700r/m and 7.6Kg mass piston on vehicle.

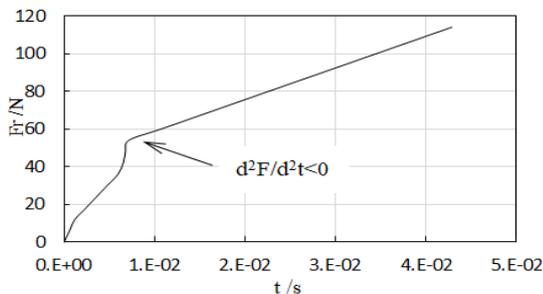


Fig-8: The relationship between force and time in crank shaft with the rotation of 700r/m and 11.4Kg mass piston on vehicle.

In short the steep force can be formed first and then form the sluggish one in a cycle meantime the big grind can be formed and them the small one. The former will damage fatigue and the later will benefit to it. The bad one will be known first and the good will be gained later in terms of former $d^2F/dt^2<0$ and $d^2F/dt^2>0$ respectively. The curve will express the summit like in Figure 6~8. Due to $d^2F/dt^2<0$ the bad linear curve has been formed.

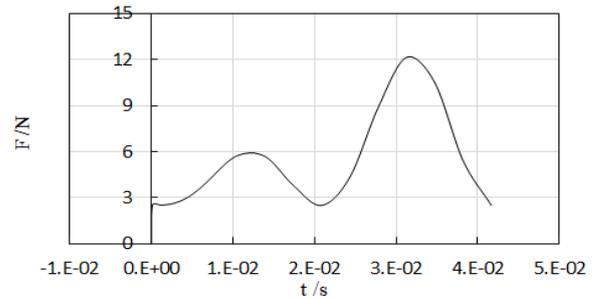


Fig-9: The relationship between force and time in crankshaft with the rotation of 720r/m and constant 0.2Kg mass piston on vehicle.

In Fig-10 the force will distribute into two sinusoidal waves. They are different with amplitude. The first one exhibits 3N whilst the second is 9N within a cycle. The force may decline as the rotation speed inclines from 12N and 11N to 10N, they exhibit the summit with 1.1E-2 and from bigger to smaller than 3E-2s in terms of Figure 9~11. the second summit may be larger than first one reaching the biggest value.

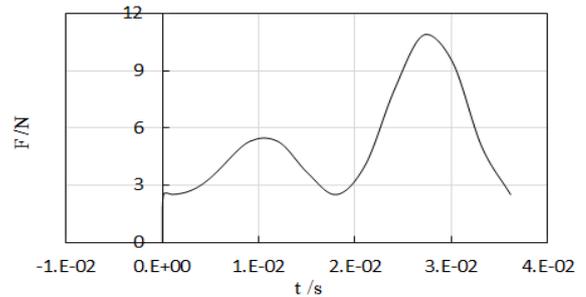


Fig-10: The relationship between force and time in crankshaft with the rotation of 830r/m and 0.2Kg mass piston on vehicle.

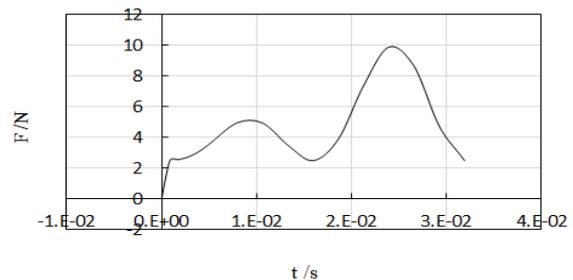


Fig-11: The relationship between force and time in crankshaft with the rotation of 940r/m and 0.2Kg mass piston on vehicle.

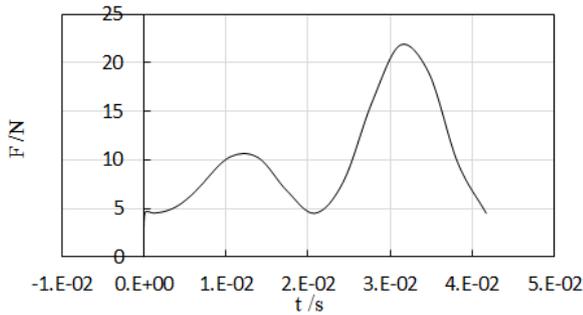


Fig-12: The relationship between force and time in crankshaft with the rotation of 940r/m and 0.45Kg mass piston on vehicle.

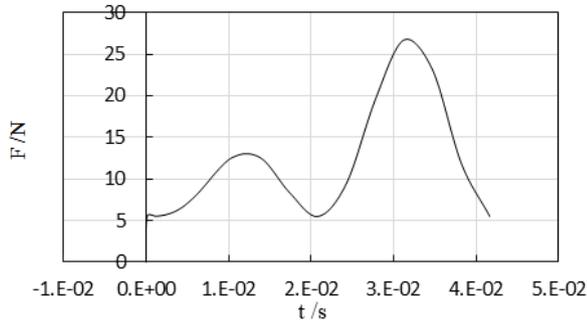


Fig-13: The relationship between force and time in crankshaft with the rotation of 940r/m and 0.55Kg mass piston on vehicle.

In Fig-12 and 13 the force may incline to 22N and 27N under the condition of 0.45Kg and 0.55kg of piston respectively. It expresses that the bigger piston mass can form the high force.

In general the force of piston may be inclined through declining the rotation and inclining its mass. In details the demanded one may be reached with big size if there are defined requirement. The second one is playing important role so it can be increased by the heavy piston.

CONCLUSIONS

With increasing time the corresponding angle will increase to 57° which is benefit to stroke stability. With the increase of time and the crank linkage angle with the force becomes big which shows sinusoidal wave. When crank and linkage length is no big relation to the force. The force will increase when the rotation increases and periodic decreases. The force attains 46N at $n=700\text{r/m}$ with $0\sim 4.6\text{Kg}$ while it attains 16N with $0\sim 1.6\text{K}$ at the same speed. If the mass of piston is increased the bigger one may be formed.

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